## Flow nozzle <br> Model : F600

## Description

The flow nozzles, more costly than other orifice due to their structure, are suited for determining the flow rates of fluids flowing at high temperature and high pressure. Under the same measuring conditions, a flow nozzle has a higher mechanical strength, can permit the flow of more than 60 percent great volume of a fluid, and can measure the flow rates of fluids containing solid particles less disturbed than an orifice having the same bore.
Thus, they are suited, in addition, for high speed flowing fluids. We can supply not only single flow nozzles, but also flow nozzles having welded short pipes on both their upstream and downstream sides.


## Specification

## Nozzle mounting types

Flange type
Weld-in type
Holding ring type
Flow calculation standards

- Long-radius flow nozzle

JIS Z 8762, IS0 5167-3, ASME MFC-3M

- ISA 1932, flow nozzle

ISO 5167-3 JIS Z 8762

## Pressure taps

1D and $1 / 2$ D tap, throat tap

## Nominal pipe sizes available


$50 ~ 630 \mathrm{~mm}$
2" ~ $25^{\prime \prime}$

## $\beta$ Limit

$0.2<\beta<0.8$
(Low - beta) long - Radius nozzle $0.2 \leq \beta \leq 0.5$
(High - beta) long - Radius nozzle $0.25 \leq \beta \leq 0.8$
$\beta:$ Ratio of throat to pipe diameter $=\mathrm{d} / \mathrm{D} 0$
(d: Throat diameter)

## Nozzle thicknesses

Maker standards

## Material



## 1. Base model

## F600 Flow nozzle

## 2. Type

w Weld in
H Holding ring
F Flanged
O Other
3. Line size

| JIS | mm | ANSI | inch | DIN | mm |
| :---: | :---: | :---: | :---: | :---: | :---: |
| J015 | 15A | A001 | $1 / 2 \mathrm{~B}$ | D015 | 15A |
| J020 | 20A | A002 | $3 / 4 \mathrm{~B}$ | D020 | 20A |
| J025 | 25A | A003 | 1B | D025 | 25A |
| J040 | 40A | A004 | 11/2B | D040 | 40A |
| J050 | 50A | A005 | 2B | D050 | 50A |
| J065 | 65A | A006 | 21/2B | D065 | 65A |
| J080 | 80A | A007 | 3B | D080 | 80A |
| J100 | 100A | A008 | 4B | D100 | 100A |
| J125 | 125A | A009 | 5B | D125 | 125A |
| J150 | 150A | A010 | 6B | D150 | 150A |
| J200 | 200A | A011 | 8B | D200 | 200A |
| J250 | 250A | A012 | 10B | D250 | 250A |
| J300 | 300A | A013 | 12B | D300 | 300A |
| J350 | 350A | A014 | 14B | D350 | 350A |
| J400 | 400A | A015 | 16B | D400 | 400A |
| J450 | 450A | A016 | 18B | D450 | 450A |
| J500 | 500A | A017 | 20B | D500 | 500A |
| J600 | 600A | A018 | 24B | D600 | 600A |
| J700 | 700A | A019 | 28B | D700 | 700A |
| J800 | 800A | A020 | 32B | D800 | 800A |
| J000 | 1,000A | A021 | 40B | D000 | 1,000A |
| xxxx | Other |  |  |  |  |

## 4. Tap type

R Radius tap
T Throat tap
5. End connection

F Flanged
w Welded on
6.Element material

4 A182 F304
5 A182 F316
6 A182 F316L
7 A182 F91
Z Other

## 7. Pipe material

| C1 | A106 Gr.B |
| :--- | :--- |
| C2 | A106 Gr.C |
| A1 | A335 P11 |
| A2 | A335 P22 |
| A3 | A335 P91 |
| XX | Other |

## 8. Holding ring material

| A1 | A182 F11 |
| :--- | :--- |
| A2 | A182 F22 |
| A3 | A182 F91 |
| C1 | A105 |
| H4 | A182 F304 |
| H5 | A182 F316 |
| ZZ | Other |
| NO | None |

9. Boss size

| 2S | $1 / 2 "$ S.W |
| :--- | :--- |
| 3S | $3 / 4^{\prime \prime}$ S.W |
| 4S | $1 "$ S.W |
| OH | Other |

## 10. Option

I Inspection pot
N None
O Other

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| F600 | w | A014 | R | w | 5 | C1 | No | 35 | N | Sample ordering code |



- Flange type -

- Weld in type -


- Throat tap type -


## Differential pressure and pressure loss

When a throttle element is interposed in a closed passage of fluid in piping, a difference is produced between the pressures upstream and downstream the throttle element as illustrated in Fig.1. This difference ( $\Delta \mathrm{P}=\mathrm{p} 1-\mathrm{p} 2$ ) is called differential pressure. The fluid passing through the section 2 gradually regains its pressure as it flows downstream, but the downstream pressure cannot be recovered up to the upstream pressure, part of the pressure being lost. This loss is called a pressure loss (permanent pressure loss = p3). The extent of this pressure loss depends on the type of throttle elements and their open area ratio, as shown in Fig. 2 The relation between the flow rate and the differential pressure is given by:
$\mathbf{Q}=\mathbf{C} \sqrt{\Delta \mathrm{P} / \rho}$
$\mathbf{Q n}=\mathbf{C} \sqrt{\Delta \mathrm{P} * \rho / \rho \mathrm{n}}$
$\mathbf{W}=\mathbf{C} \sqrt{\Delta \mathbf{P} * \rho}$
$\mathrm{Q}\left(\mathrm{m}^{3} / \mathrm{h}\right)$ : Volume rate of flow at density operating conditions
Qn ( $\mathrm{Nm}^{3} / \mathrm{h}$ ) : Volume rate of flow at density bass conditions
W (kg/h) : Weight rate of flow
$\rho\left(\mathrm{kg} / \mathrm{m}^{3}\right)$ : Density in operating conditions
$\rho n\left(\mathrm{~kg} / \mathrm{Nm}^{3}\right)$ : Density in base conditions
C : Constant coefficient

From the above, the relation between the flow rate and the differential pressure where the density is constant but the flow rate is variable is as listed in table 1. In other words, the flow rate is obtainable by measuring the differential pressure. When the density is variable (When the pressure and temperature are variable), the true flow rate can be given by compensating the variate of the density by the above equation (This however, is not applicable when the density varies to a great extent.)


Fig. 1

Table 1 : Relation between Flow Rate and Differential Pressure



Fig. 2

